

REMARKS

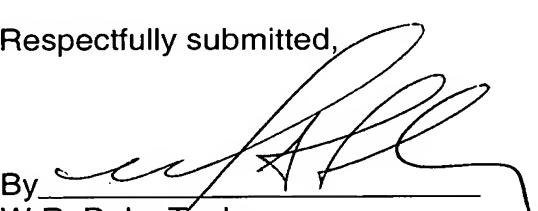
Claims 1 through 5 have been amended and remain pending in the present application. In view of the above amendment, applicant believes the pending application is in condition for allowance.

Thus, prompt and favorable consideration of this amendment is respectfully requested. If the Examiner believes that personal communication will expedite prosecution of this application, the Examiner is invited to telephone the undersigned at (248) 641-1600.

Applicant believes no fee is due with this response. However, if a fee is due, please charge our Deposit Account No. 08-0750, under Order No. 6340-000071/US/NP from which the undersigned is authorized to draw.

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Respectfully submitted,

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SEALED ROLLING BEARING

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This application is a National Stage of International Application No. PCT/JP2004/015711, filed October 22, 2004, which claims priority to Japanese Patent Application No. 2003-367307, filed October 28, 2003 and Japanese Patent Application No. 2004-004245, filed January 9, 2004. The disclosures of the above applications are incorporated herein by reference.

FIELD

[0002] [0001] The present invention relates to a sealed rolling bearing used ~~for in~~ in automobiles, motors in general use etc., and, more particularly, to a sealed roller bearing used under circumstances ~~in which~~where a great deal of water, muddy water and many other foreign matters exist.

BACKGROUND

[0003] [0002] In general, sealed roller bearings, having with a high sealability, preventing ingress of rain water or dusts into the inside of the bearing. The bearings are used, for example, as bearings for the suspension of automobiles. This is due to the fact that since these bearings are usually exposed to severe conditionscircumstances which containingcontain muddy water, dusts etc. On the other hand, such a rolling bearing is required to have its a small rotational torque since a large rotational torque ~~of on~~on the bearing causes adverse influence to bearing temperature rise ~~of the bearing~~ and fuel consumption. Since the sliding resistance of the seals is a major factor ~~of that~~ influencescausing an increase ~~of in~~ in the rotational

torque of the bearing, it ~~has been desired~~ is ~~desirous~~ to provide a sealed rolling bearing ~~that has~~ having not only a high sealability but a small sliding resistance.

[0004] [0003] One representative example of such a bearing for an automobile ~~will be~~ is described with reference to Fig. 1. Fig. 1 ~~showing~~ shows a first embodiment of the present ~~invention~~ and disclosure. This bearing is ~~that~~ used for a driving wheel of an automobile. The bearing includes and comprises an outer member 4 integrally formed with a body mounting flange on its outer circumference, ~~with a body mounting flange~~ 2. The flange is adapted to be mounted on a body (not shown) of the automobile. The outer member is also and ~~also~~ formed on its inner circumferential surface with double row outer raceway surfaces on its inner circumferential surface. A wheel hub 8 and 8; a hub wheel 4 has an integrally formed wheel mounting flange on its one of its end to mount with a wheel mounting flange 7 on which a wheel (not shown). One inner raceway surface is mounted and also formed on its the outer circumferential surface of the wheel hub. with one inner raceway surface 9a. The inner raceway surface is arranged oppositely to one of the double row outer raceway surfaces. A 8 and 8 and having a cylindrical portion 14 of a smaller diameter, axially extending extends from the inner raceway surface 9a and further. The cylinder portion includes serrations formed on its inner circumferential surface with a serration 6 to transmit for torque. An transmission; and an inner ring 5 is adapted to be fitted fitted on the cylindrical portion 14 and and the inner ring includes the other inner raceway surfaces formed on its outer circumferential surface with the other inner raceway surface 9b.

[0005] [0004] Double row rolling elements (balls) 10 are arranged between the double row outer and inner raceway surfaces. The balls are 8 and 8; 9a and 9b

and held freely rollably held by cages 11. Sealing devices 12 and 13 are arranged at both ends of the outer member 1 within an annular space formed by an inner member 3 (including the hub-wheel hub 4 and the inner ring 5) and the outer member 1. The sealing devices to prevent leakage of grease contained within the bearing as well as ingress of rain water or dusts into the inside of the bearing.

[0006] [0005] As shown in Fig. 2, the The sealing device of the 12 of inboard side, arranged between the outer member 1 and the inner ring 5, comprises includes a sealing ring 17 including with a metal core. The metal core has 15 having a substantially L-shaped cross-section and is fitted into the outer member 1, and a A sealing member 16 is integ ally adhered, via vulcanized adhesion, on onto the metal core 15. A, and a slinger 18, having a similar similar a L-shaped cross-section, and is fitted onto the inner ring 5. The sealing member 16 is made of an elastic material, such as rubber, and has comprises three sealing lips, i.e. an outer sealing lip 23, a middle sealing lip 24 and an inner sealing lip 25. The tip edge of the outer sealing lip 23 is in sliding contacted with to an inner side of an upstanding portion 22 of the slinger. The 18 and the tip edges of the middle sealing lip 24 and the inner sealing lip 25 are in sliding contacted with to a cylindrical portion 21 of the slinger 18.

[0007] [0006] On the other hand the The sealing device 13, on the outboard side, of outboard side comprises as shown in Fig. 3 has an annular metal core 26 and a sealing member 27 integ ally adhered, via vulcanized adhesion, on the metal core 26. The sealing member 27 is made of elastic material, such as rubber, and has comprises three sealing lips, 27a, 27b and 27c and the tip edges of these the

three sealing lips 27a, 27b and 27c are in directlydirect sealing contacted contacte with the surface of the hub-wheel hub4.

[0008] {0008} In thesethe sealing devices 12 and 13 of the bearing for a wheel of a vehicle of the prior art, the slinger 18 of the sealing device 12on the of inboard side has thea surface roughness of a sliding surface toon which the sealing lips 23, 24 and 25 sliding contact which is limited to a value at the center line average height (Ra) at 0.3 μ m or less and to a value at the maximum height (Ry) at 1.2 μ m or less. Thus, foreign materials cannot easily enter into the inside of the bearing. This is due to the because of difficulty ofto generationgenerate ofa small gap at the maximum height portions and damages which would be otherwise caused at rolling contact portions by change ofin the properties of the grease (see Japanese Laid-open Patent Publication No. 184897/2003).

[0009] {0008} However, in order to limit the surface roughness of a sliding surface to which the sealing lips 23, 24 and 25 sliding contact to the value at the center line average height (Ra) at 0.3 μ m or less and to the value at the maximum height (Ry) at 1.2 μ m or less, it is necessary, previously, to press sheet members having the target surface roughness or to carry out lapping of the surfaces of sheet members after the pressing process thereof. In fact, since availability of the plate member with having such a target surface roughness is difficult, it is the state of artthings, in view of its manufacturing cost, to carry out lapping of the surfaces of the sheet members after the pressing process thereof.

[0010] {0009} Such a lapping of the surface of the slinger 18 makes its handling very difficult, increases the number of processing steps, and further causes deformation of the surface of the slinger 18. Such a deformation of the sliding

surface of the slinger 18-causes variation of the interference of the sealing member. This reduces and thus reduction of the followability following ability of the sealing lips and accordingly reduces the sealing sealability of the bearing.

SUMMARY

[0011] [0010] It is, therefore, an object of the present disclosure invention to provide a sealed rolling bearing provided with sealing devices which provide having both functions of high sealability and small sliding resistance, which are antipodal to each other.

[0012] [0011] For achieving To achieve the object mentioned above, there is provided, according to the present invention of claim 1, a sealed rolling bearing comprising comprises an outer member with outer raceway surfaces formed on its inner circumferential surface. An with outer raceway surface; an inner member is formed with inner raceway surfaces formed on its outer circumferential surface. The with inner raceway surface is arranged oppositely opposite to the outer raceway surface. Rolling; rolling elements are contained freely rollably contained between the outer and inner raceway surfaces. ; and sealing Sealing devices are arranged in an annular space formed between the outer and inner members. characterized in that each Each of the sealing devices has sealing lips formed from an of elastic member. ; that the The maximum height Ry or Rmax of the surface roughness of a sliding surface of a member of the rotational side to which the sealing lips sliding contact is limited to a value at of 2.0µm or less. The, and that the run-out of the sliding surface, normal thereto, is limited to a value at of 30µm or less.

[0013] [0012] According to the present invention of claim 1, since Since each of the sealing devices has sealing lips formed from an of elastic member and; that

the maximum height R_y or R_{max} of the surface roughness of ~~a~~the sliding surface of a member of the rotational side ~~to~~ which the sealing lips sliding contact is limited to a value of at 2.0 μm or less, and ~~that~~ the run-out of the sliding surface normal thereto is limited to a value at of 30 μm or less, it is possible to suppress the absolute irregularities of the sliding surface to a small as well as variation. Also, it is possible to suppress variation of the interference without increasing the interference of the sealing member. This and thus to avoid avoids the problem of deformation of the sliding surface which is caused by the lapping of the slinger. Accordingly, it is possible to stabilize the followabilityfollowing ability of the sealing lips relative to the sliding surface and thus further ~~to~~ improve the sealability.

[0014] ~~[0013]~~ It is preferable, ~~as defined in claim 2~~, that the sealing device ~~comprises~~includes a sealing ring mounted on a member of the stationary side and a slinger mounted on a member of the rotational side. ~~The, and that the sealing lips, forming the sealing ring, are in sliding contacted~~contact with ~~to~~ the slinger. This structure makes it unnecessary to strictly restrict the target surface roughness by applying a lapping process to the sliding surface after the pressing process. Thus, it ~~and~~ it is possible to improve the sealability only by only setting the run-out of the sliding surface at a predetermined value.

[0015] ~~[0014]~~ Also, ~~it is preferable, as defined in claim 3~~, the sealing device includes ~~comprises~~ a sealing ring mounted on a member of the stationary side, and ~~including~~ side lips and a radial lip. ~~The, and that the sealing lips are in directly~~direct sliding contact with ~~to~~ the member of the rotational side. This structure makes it possible to easily carry out ~~the~~ surface machining, such as grinding or

lapping, of the sliding surface after heat treatment thereof even if the target surface roughness and the run-out are would not be obtained.

[0016] [0015] Further, it is preferable, as defined in claim 4, that the sealing device includes comprises a sealing ring mounted on a member of the stationary side, and including a main lip and a sub lip. The that the main lip is in directly direct sliding contacted contact with to a sealing groove formed on a member of the rotational side. and having The sealing groove has a substantially U-shaped cross-section, and that the The sub lip is in sliding contacted contact with to a ridge of the sealing groove, via a small interference. This structure makes it possible to increase the interference, with while keeping the sealability sealing ability, due to ability of the following ability of the sub lip when the main lip wears. Accordingly, it is possible to suppress the rotational torque of the bearing during a small amount of wear of the main lip is kept small as well as to obtain the sealability sealing ability due to an increase of interference in accordance with the wear of the main lip.

[0017] [0016] It is also preferable, as defined in claim 5, that Also, preferably the maximum height Ry or Rmax of the surface roughness of the sliding surface is limited to a value of at 1.2 μ m or less. The, and the run-out of the sliding surface, normal thereto, is limited to a value at of 10 μ m or less. This enables to provide provides a sealed rolling bearing provided with sealing devices which has having both functions of high sealability sealing ability and small sliding resistance, which are antipodal opposite to each other.

[0018] [0017] According to the present invention, the The sealed rolling bearing comprises includes an outer member formed with outer raceway surface on its inner circumferential surface with outer raceway surface. An; an inner member is

formed with an inner raceway surface on its outer circumferential surface. The inner raceway surface ~~is with inner raceway surface~~ arranged oppositely opposite to the outer raceway surface. Rolling; rolling elements are contained freely rollably contained between the outer and inner raceway surfaces; and Sealing sealing devices are arranged in an annular space formed between the outer and inner members. Each and ~~is characterized in that~~ each of the sealing devices has sealing lips of elastic member sealing lips. The, ~~that~~ the maximum height R_y or R_{max} of the surface roughness of a sliding surface of a member ~~efon~~ on the rotational side, to which the sealing lips sliding contact, is limited to a value at of $2.0\mu m$ or less. The, ~~and that~~ the run-out of the sliding surface, normal thereto, is limited to a value at of $30\mu m$ or less. This makes it possible to suppress the absolute irregularities of the sliding surface to a small variation as well as to suppress variation of the interference without increasing the interference of the sealing member. Thus, this ~~and thus~~ avoids ~~to avoid~~ the problem of deformation of the sliding surface which is caused by the lapping of the slinger. Accordingly, it is possible to stabilize the fellowability following ability of the sealing lips relative to the sliding surface and thus ~~to~~ further to improve its ~~the~~ sealability sealing ability.

[0019] [0018] ~~A~~ The best mode for carrying out the present invention is a sealed rolling bearing includes comprising an outer member with outer raceway surface formed on its inner circumferential surface ~~with outer raceway surface~~; an inner member formed with inner raceway surface on its outer circumferential surface ~~with inner raceway surface~~ arranged oppositely opposite to the outer raceway surface; rolling elements contained freely rollably contained between the outer and inner raceway surfaces; and sealing devices arranged in an annular space formed

between the outer and inner members. Each characterized in that each of the sealing devices has elastic member sealing lips. The of elastic member; that the maximum height Ry or Rmax of the surface roughness of a sliding surface of a member of the rotational side, to which the sealing lips sliding contact, is limited to a value of at 2.0 μ m or less. The, and that the run-out of the sliding surface, normal thereto is limited to a value at of 30 μ m or less.

[0020] [0019] Further areas of applicability will become apparent from the description provided herein. It should be understood that the description and specific examples are intended for purposes of illustration only and are not intended to limit the scope of the present disclosure.

DRAWINGS

[0021] [0029] Additional advantages and features of the present invention disclosure will become apparent from the subsequent description and the appended claims, taken in conjunction with the accompanying drawings, wherein:

[0022] [0021] Fig. 1 is a longitudinal-section view of a first embodiment of a sealed rolling bearing according to a first embodiment, of the present invention;

[0023] [0022] Fig. 2 is a partially enlarged sectional view of the sealing device of the inboard side of the sealed rolling bearing according to the first embodiment of the present invention;

[0024] [0023] Fig. 3 is a partially enlarged sectional view of the sealing device of the outboard side of the sealed rolling bearing according to the first embodiment of the present invention;

[0025] [0024] Fig. 4 is a longitudinal-section view of a sealed rolling bearing according to a second embodiment of the present invention;

[0026] [0025] Fig. 5 is a partially enlarged sectional view of a sealing device of a second embodiment of the present invention; and

[0027] [0026] Fig. 6 is a graph showing results of mass variation in a muddy water proof test carried out by exposing a naked bearing, itself, to the muddy water.

DETAILED DESCRIPTION

[0028] [0027] Preferred embodiments of the present invention will be described with reference to accompanied drawings.

[0029] [0028] Fig. 1 is a longitudinal-section view of a first embodiment of a sealed roller bearing of the present invention. The sealed rolling bearing shown in Fig. 1 is an example applied to a driving wheel of a vehicle for rotatably supporting the wheel in a suspension of the vehicle. Since the fundamental structure of the bearing provided with the sealing devices previously mentioned has been described at the beginning of this specification using this Fig. 1, characteristic portions of the bearing will be mainly described hereinafter.

[0030] [0029] Sealing devices 12 and 13 are Arranged arranged at either ends of an annular space formed between the outer member 1 and inner member 3, comprised of comprising the hub-wheel hub 4 and the inner ring 5 and which form forming a member of the rotational side. The and the outer member 1 forming forms a member of the stationary side. The are the sealing devices 12 and 13 for preventing prevent leakage of grease contained within the bearing as well as ingress of rain water or dusts into the inside of the bearing.

[0031] [0030] The sealing device 12 ~~mounted~~mounts between the outer member 1 and the inner wheel 5 ~~at~~on the inboard side (the right hand side in Fig. 1). The sealing device 12 ~~comprises~~, as shown in Fig. 2, includes a sealing ring 17 ~~within~~including a metal core 15. The metal core 15 ~~has~~ having a substantially L-shaped cross-section. The sealing device 12 ~~is~~ and fitted into the outer member 1. A ~~and~~ a sealing member 16 is integrally adhered, via vulcanized adhesion, ~~on~~onto the metal core 15. A ~~and~~ a slinger 18, ~~with a similar~~ having similarly a L-shaped cross-section, is fit ~~and~~ fitted onto the inner ring 5. The slinger 18 and the metal core 15 are made by press forming of austenitic stainless steel sheet (JIS SUS 304 etc.) or preserved cold rolled sheet (JIS SPCC etc.).

[0032] [0031] The sealing member 16 is made of an elastic material such as rubber and ~~includes~~comprises three sealing lips, *i.e.* an outer sealing lip 23, a middle sealing lip 24 and an inner sealing lip 25. The tip edge of the outer sealing lip 23 ~~is~~ sliding contacted ~~contacts~~ to an inner side of an upstanding portion 22 of the slinger 18. The ~~and~~ the tip edges of the middle sealing lip 24 and the inner sealing lip 25 are in ~~sliding contacted~~ contact with ~~to~~ a cylindrical portion 21 of the slinger 18. The surface roughness of a sliding surface of the slinger 18, ~~to~~ which the sealing lips 23, 24 and 25 are in ~~sliding contact with~~ is limited to a value at ~~of~~ $2.0\mu\text{m}$ or less, and preferably at $1.2\mu\text{m}$ or less at the maximum height Ry or R_{max} . The ~~and~~ the run-out of the sliding surface, ~~normal thereto~~, is limited to a value at ~~of~~ $30\mu\text{m}$ or less, and preferably at $10\mu\text{m}$ or less. That is Accordingly, it is unnecessary to strictly restrict the target surface roughness by applying a lapping process to the sliding surface after the pressing process as in the prior art. This ~~and~~ this is achieved only by setting the run-out of the sliding surface at a predetermined value.

[0033] [0032] Thus, it is possible to suppress the absolute irregularities of the sliding surface to a small variation as well as to suppress variation of the interference without increasing the interference of the sealing member. Thus, this avoids and thus to avoid the problem of deformation of the sliding surface of the slinger 18 which is caused by the lapping the of the slinger as in the prior art. Accordingly, it is possible to stabilize the followabilityfollowing ability of the sealing lips 23, 24 and 25 relative to the sliding surface and to thus further to improve the sealability.

[0034] [0033] On the other hand the The sealing device 13 on the outboard side, comprises as shown in Fig. 3, includes an annular metal core 26 and a sealing member 27 integrally adhered, via vulcanized adhesion, on the metal core 26. The metal core 26 is formedmade by press forming of austenitic stainless steel sheet (JIS SUS 303 etc.) or preserved cold rolled sheet (JIS SPCC etc.). The sealing member 27 is made of an elastic material, such as rubber, and includes comprises two side lips (dust seal) 27a, 27b and one radial lip (grease seal) 27c. The, and the tip edges of the sealing lips 27a~27c are in directlydirect sliding contactedcontact with to the surface of the hub-wheel hub 4. The tip edges contact, that is, a sliding surface 19 on the base of the wheel mounting flange 7 aton the inboard side.

[0035] [0034] The maximum height Ry or Rmax of the surface roughness of the a sliding surface 19, to which slidingly contacts the sealing lips 27a, 27b and 27c, sliding contact is limited to a value atof 2.0 μ m or less, and preferably at 1.2 μ m or less. The and that the run-out of the sliding surface 19, normal thereto, is limited to a value atof 30 μ m or less, and preferably at 10 μ m or less. If these target surface roughness or run-out cannot be obtained, a grinding or lapping process may be applied toconducted on the sliding surface after itsit is heat treatmenttreated.

[0036] [0035] Thus, ~~similarly~~similar to the sealing device 12, it is possible to suppress the absolute irregularities of the sliding surface 19 to a small variation as well as to suppress variation of the interference without increasing the interference of the sealing member. ~~This avoids and thus to avoid~~ the problem of deformation of the sliding surface. Accordingly, it is possible to stabilize the ~~fellowability~~following ability of the sealing lips 27a, 27b and 27c relative to the sliding surface 19 and to thus further ~~to improve the its sealability sealing ability~~.

[0037] [0036] Fig. 4 is a longitudinal-section view of a sealed rolling bearing according to a second embodiment of the present ~~invention~~disclosure. Fig. 5 is a partially enlarged sectional view of Fig. 4. The sealed rolling bearing 20, exemplary shown in Fig. 4, is a deep groove ball bearing. ~~An and comprises an outer ring 30 is formed with an outer raceway surface 29 on its inner circumferential surface. An with an outer raceway surface 29, an inner ring 32 is formed with an inner raceway surface 31 on its outer circumferential surface. Balls with an inner raceway surface 31, balls 34 are contained and held freely rollably held by a cage 33 between the outer and inner raceway surfaces 29 and 31. A, and a pair of sealing rings 35 are arranged in an annular space formed between the outer and inner rings 30 and 32.~~

[0038] [0037] Each of the sealing ~~ring~~rings 35 ~~has~~ comprises a metal core 36 ~~with~~of flat annular configuration formed ~~off~~from cold rolled steel sheet (JIS SPCC etc.) by press forming. ~~A and a~~ sealing member 37 is integrally adhered, via vulcanized adhesion, ~~on~~onto the metal core 36. The sealing rings 35 are ~~fitted into~~into the inner circumferential surface at both ends of the outer ring 30, via the sealing members 37. Each of the sealing ~~member~~members 37 is ~~in direct~~directly sliding ~~contacted~~contact with a ~~to a~~ sealing groove 38. ~~The sealing groove 38 has~~ having a

substantially U-shaped cross-section formed at each end on the outer circumferential surface of the inner ring at each end thereof. AMore in detail the a main lip 37a of the sealing member 37 contacts an inclined sliding surface 39 of the sealing groove 38. A and a sub lip 37b of the sealing member 37 a contacts a ridge 40 of the sealing groove 38, via a small interface therebetween. The base of the sub lip 37b has a constricted cross-section. Thus, and thus the sub lip 37b has a small flexural rigidity. Accordingly, the sub lip 37b can follow the motion of the main lip 37a and move toward the left hand direction (Fig. 5) in accordance with the wear of the main lip 37a. Thus, this increases and thus increase the interference relative to the ridge 40. Thus the sub lip 37b can suppress the rotational torque of the bearing during thea small wearing amount of wear of the main lip 37a is small and can on the other hand ensure its sealability sealing ability while with increasing the interference relative to the ridge 40 in accordance with wear of the main lip 37a.

[0039] [0038] The maximum height Ry or Rmax of the surface roughness of a at least the sliding surface 39 of the sealing groove 38, to which the main sealing lip 37a slidingslidingly contacts, is limited to a value at of 2.0 μ m or less, and preferably at 1.2 μ m or less. The, and that the run-out of the sliding surface 39, normal thereto, is limited to a value at of 30 μ m or less, and preferably at 10 μ m or less. If these target surface roughness or run-out cannot be obtained, a grinding or lapping process may be conducted on applied to the sliding surface after its it is heat treatment treated.

[0040] [0039] Thus, it is possible to suppress the absolute irregularities of the sliding surface 39 to a small variation as well as to suppress variation of the interference without increasing the interference of the sealing member. Thus, this stabilizes and thus to stabilize the followabilityfollowing ability of the sealing lip 37a

relative to the sliding surface 39 and ~~thus further to improve~~improves its the sealing ability~~sealability~~.

[0041] ~~[0040]~~ Fig. 6 is a graph showing results of mass variation ~~of muddy water~~ in a muddy water proof test carried out by exposing a naked bearing, itself, to the muddy water. This test was carried out by spraying Kanto team ~~foam~~ JIS 8 mixed liquor onto samples of sealed rolling bearings of the present disclosure~~invention~~ ~~during the driving and the prior art during driving~~, and then by measuring the mass variation before and after the test. As can be seen from Fig. 6, there are remarkable differences ~~among~~between sample 1 (Comparative example) having the maximum height Ry or Rmax of 2.02~3.7 μ m and the run-out of 30 μ m normal to the sliding surface 39, a sample 2 (Embodiment A) having the maximum height Ry or Rmax of 1.3~1.86 μ m and the run-out of 10~30 μ m normal to the sliding surface 39, and a sample 3 (Embodiment B) having the maximum height Ry or Rmax of 0.7~1.2 μ m and the run-out of 10 μ m or less normal to the sliding surface 39.

[0042] ~~[0041]~~ The sealed rolling bearing of the present invention~~disclosure~~ can be applied to any type of sealed rolling bearings, irrespective of its bearing or sealing type, used under circumstances ~~in which~~where a large amount of foreign matter, such as water or muddy water, exists.

[0043] ~~[0042]~~ The present disclosure~~invention~~ has been described with reference to the preferred embodiment. Obviously, modifications and alternations will occur to those of ordinary skill in the art upon reading and understanding the preceding detailed description. It is intended that the present invention be construed to as including~~including~~ all such alternations and modifications insofar as they come within the scope of the appended claims or ~~the~~their equivalents ~~thereof~~.

CLAIMS

What is claimed is:

1. A sealed rolling bearing comprising:

an outer member (1; 30) formed with at least one outer raceway surface on its inner circumferential surface;

with outer raceway surface (8; 29); an inner member (4, 5; 32) formed with at least one inner raceway surface on its outer circumferential surface, said inner raceway surface with inner raceway surface (9a, 9b; 31) arranged oppositeoppositely to the outer raceway surface;

(8; 29); rolling elements (10; 34) contained freely rollably contained between the outer and inner raceway surfaces; and

sealing devices (12, 13; 35) arranged in an annular space formed between the outer and inner members (1; 30 and 4,5; 32) characterized in that:

each of the sealing devices (12, 13; 35) has sealing lips (27a~27c, 23~25; 37a, 37b) of an elastic member; that the maximum height Ry or Rmax of the surface roughness of a sliding surface of a member of a rotational side (18, 4; 32) to which the sealing lips (27a~27c, 23~25; 37a, 37b) sliding contact, is limited to a value atof 2.0 μ m or less, and that the run-out of the sliding surface, normal thereto, is limited to a value ofat 30 μ m or less.

2. A-The sealed rolling bearing of claim 1 wherein the sealing device includes (12) comprises aa sealing ring (17 or 15, 16) mounted on a member (1) of a stationary side and a slinger (18) mounted on a member (5) of a rotational side, and

~~wherein the sealing lips (23~25) forming the sealing ring (17 or 15, 16) are sliding contact contacted to the slinger (18).~~

3. ~~The~~A sealed rolling bearing of claim 1 wherein the sealing device (13) includes~~comprises~~ a sealing ring, (26, 27) mounted on a member of a stationary side, with~~and including~~ side lips (27a, 27b) and a radial lip (27c), and ~~wherein~~ the sealing lips (27a~27c) are directly ~~slidingly~~ ~~in~~ ~~contact~~ ~~slidely~~ ~~contacted~~ to the member of a rotational side.

4. ~~The~~A sealed rolling bearing of claim 1 wherein the sealing device includes~~comprises~~ a sealing ring (35), mounted on a member of a stationary side, with~~(30)~~ and including a main lip (37a) and a sub lip (37b), ~~wherein~~ the main lip (37a) is directly sliding contacted~~contacts~~ to a sealing groove (38) formed on a member of a rotational side, said sealing groove (32) and having a substantially U-shaped cross-section, and ~~wherein~~ the sub lip (37a) is slidely contacted~~contacts~~ to a ridge (40) of the sealing groove (38) via a small interference.

5. ~~The~~A sealed rolling bearing of any one of claims 1~4 wherein the maximum height Ry or Rmax of the surface roughness of the sliding surface is limited to a value at~~of~~ 1.2 μ m or less, and the run-out of the sliding surface, normal thereto, is limited to a value of~~at~~ 10 μ m or less.

ABSTRACT

An object of the present invention is to provide a sealed rolling bearing ~~has provided~~ with sealing devices ~~with~~ having both the functions of high sealability sealing ability and small sliding resistance, which are antipodal to each other. According to the present invention, there is provided, a The sealed rolling bearing ~~has comprising~~ an outer member (1, 30) formed ~~with~~ an outer raceway surface (8, 29) on its inner circumferential surface. An ~~with~~ outer raceway surface (8, 29); an inner member (4, 5, 324, 5; 32) is formed ~~with~~ an inner raceway surface (9a, 9b, 31) on its outer circumferential surface. The inner raceway surface (9a, 9b, 31) is ~~with~~ inner raceway surface (9a, 9b; 31) arranged oppositely opposite to the outer raceway surface (8, 298; 29). Rolling; rolling elements (10, 34) are contained freely rollably ~~contained~~ between the outer and inner raceway surfaces. Sealing; and sealing devices (12, 13, 35) are arranged in an annular space formed between the outer and inner members (1, 301; 30 and 4, 5, 324, 5; 32). Each characterized in that: each of the sealing devices (12, 13; 35) has elastic sealing lips (27a~27c, 23~25, 37a, 37b). The of elastic member; that the maximum height Ry or Rmax of the surface roughness of a sliding surface of a member of the rotational side (18, 4, 3218, 4; 32), to which the sealing lips (27a~27c, 23~25, 37a, 37b) ~~slidingly~~ contact, is limited to a value at of 2.0 μ m or less. The, and that the run-out of the sliding surface, normal thereto, is limited to a value of at 30 μ m or less.